

# Pressurisation in microstructured optical fibre drawing

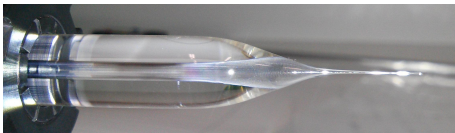
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Herbert Foo<sup>††</sup>, Alastair Dowler<sup>††</sup> and Heike Ebendorff-Heidepriem<sup>††</sup>

<sup>\*</sup>University of Oxford

<sup>\*\*</sup>The University of Adelaide

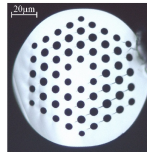
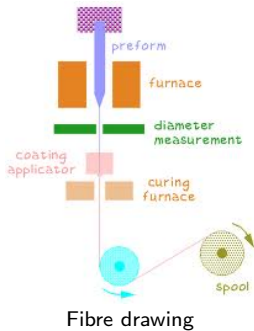
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<sup>††</sup>IPAS, Adelaide



# Microstructured optical fibres (MOFs)

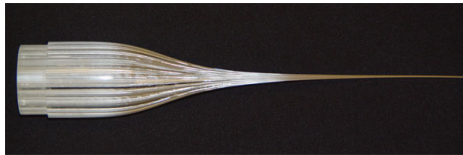
Have a large number of wavelength-scale channels which make for novel optical properties.



Cross section

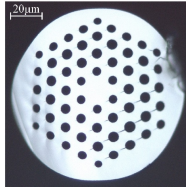
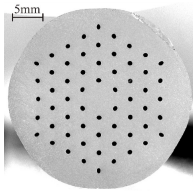
Fabricated by **drawing** a heated **preform** onto a rotating spool.

$$\text{Draw ratio} = \frac{\text{draw speed}}{\text{feed speed}} \gg 1.$$



Photos: IPAS, Adelaide

# Fibre drawing modifies the geometry



During fibre drawing the cross section deforms under surface tension and channel pressurisation.

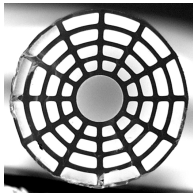
**Goal: understand this deformation.**

Forward problem:

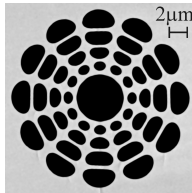
For given preform geometry and draw parameters ... what fibre geometry?

Inverse problem:

For a desired fibre geometry ... what preform geometry and draw parameters?



preform



fibre

## Mathematical model

- ▶ Consider: surface tension, pressurisation of channels, varying viscosity.
- ▶ Neglect: inertia, gravity.

# Model with internal pressure

3D steady Stokes' flow; slender geometry ( $\epsilon = \sqrt{S_0}/L$ ).

## Axial model

$$\chi^2(\tau) u(\tau) = 1$$

$$\frac{d\chi}{d\tau} - \frac{\chi}{12} \tilde{\Gamma}(\tau) = -\mathcal{T}$$

- ▶  $\tau$  is a Lagrangian reduced time co-ordinate,  $\tau = \gamma^* \int_0^t \frac{dt}{\mu^* \chi}$ ;
- ▶  $\chi^2(\tau)$  is the cross-sectional area;
- ▶  $\tilde{\Gamma}(\tau)$  is the scaled total circumference.

If there's axially varying pressure in the channels

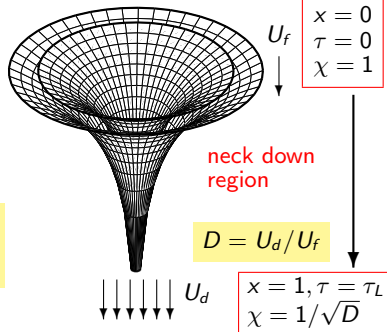
$$(3\mu^* S u_x)_x + \frac{1}{2} \gamma^* \Gamma_x + \gamma^* \sum_{k=1}^N \mathcal{P}_x^{(k)} A^{(k)} = 0$$

## Cross-plane model

$$\nabla \cdot U = 0, \quad -\nabla p + \nabla^2 U = 0 \quad \text{in fluid}$$

$$\sigma \cdot \hat{n} = -\left(\kappa + \mathcal{P}^{(k)} \chi\right) \hat{n} \quad \text{on surface}$$

$$0 = G_\tau + v G_y + w G_z \quad \text{on surface}$$



## Model with internal pressure

3D steady Stokes' flow; slender geometry ( $\epsilon = \sqrt{S_0}/L$ ).

### Axial model

$$\chi^2(\tau) u(\tau) = 1$$

$$\frac{d\chi}{d\tau} - \boxed{\frac{\chi}{12} \tilde{\Gamma}(\tau)} = -\mathcal{T}$$

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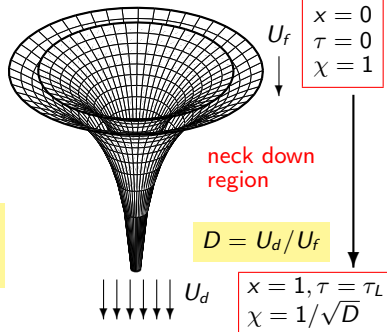
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$$0 = G_\tau + \nu G_y + w G_z \quad \text{on surface}$$

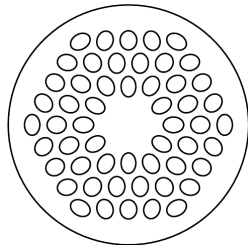


## Solving the 2D cross-plane model

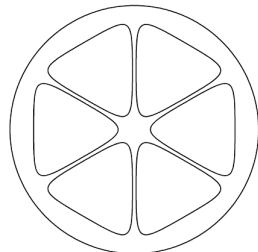
Straightforward for the fundamental case of a tubular fibre.

For more general geometries have a suite of numerical solution techniques based on complex variable methods:

- ▶ Elliptical pore model: channel shape are ellipses (Buchak et al, JFM '15)

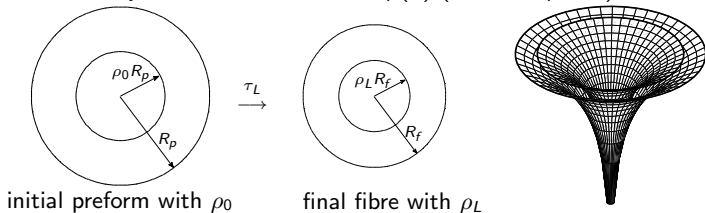


- ▶ Spectral method: more general channel shapes (Buchak & Crowdy, JCP '16)



## Tubular fibres

The two boundaries are constrained to be concentric circles - it's convenient to describe them by the ratio of their radii,  $\rho(\tau)$  (with  $0 < \rho < 1$ ).



The governing equations, which are solved from  $\chi = 1$  to  $1/\sqrt{D}$ , are:

$$\begin{aligned}\frac{d\chi}{d\tau} &= \frac{\sqrt{\pi}}{6} \chi (1 + \rho)^{\frac{1}{2}} (1 - \rho)^{-\frac{1}{2}} - \mathcal{T}, \\ \frac{d\rho}{d\tau} &= -\frac{\sqrt{\pi}}{2} (1 + \rho)^{\frac{3}{2}} (1 - \rho)^{\frac{1}{2}} + \frac{1}{2} \mathcal{P} \rho \chi.\end{aligned}$$

Additionally, it is sometimes useful to solve for the axial position  $x$ :

$$\frac{dx}{d\tau} = \frac{\mu^*}{\gamma^* \chi}$$

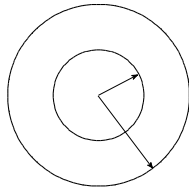
## Tubular fibres: summary of experiments

Six extruded tubular preforms were drawn to fibre.

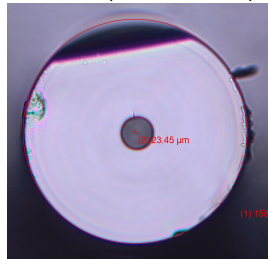
- ▶ Account for taper from extrusion process.
- ▶ Initial geometry either  $\rho_0 = 0.16$  or  $\rho_0 = 0.515$ .
- ▶ Systematically vary one operational parameter at a time: furnace temperature (and therefore tension), draw speed or pressurisation.
- ▶ Take microscope measurements of fibre outer and inner diameter from the drum.



Preform: 10mm OD with  $\rho_0$

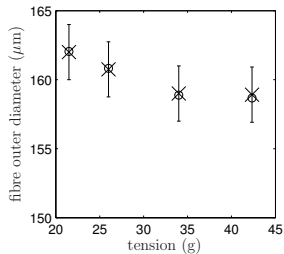


Fibre: 160 $\mu$ m OD with  $\rho_L$

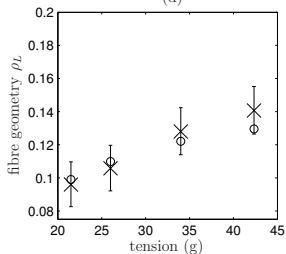


# Results

Experiment 2

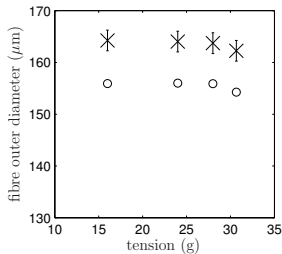


(d)

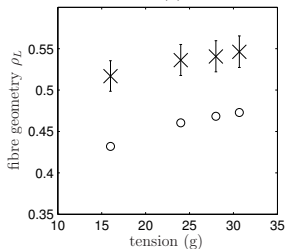


$\rho_0 = 0.16$ ,  $U_f = 1.4\text{mm/min}$ ,  
 $U_d = 6.1\text{m/min}$ ,  $T_f = 940\text{--}905^\circ\text{C}$

Experiment 4



(d)

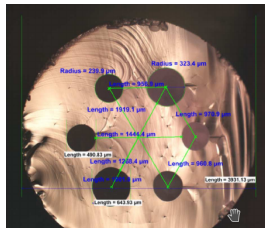


$\rho_0 = 0.515$ ,  $U_f = 1.4\text{mm/min}$ ,  
 $U_d = 5.9\text{m/min}$ ,  $T_f = 980\text{--}950^\circ\text{C}$

# Multi-hole preforms: two experiments

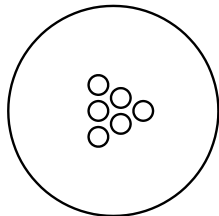
## 1. From the literature (Luzi et al. JLT 2012):

- ▶ Six holed preform, drawn with a range of (quite large) pressurisations.
- ▶ Relative small outer diameter (3.93 mm), small draw ratio ( $D=935$ )

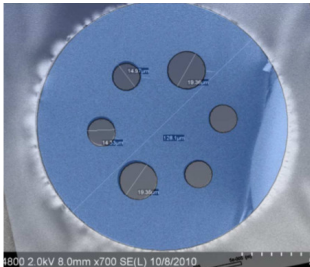


## 2. A new purpose-designed experiment:

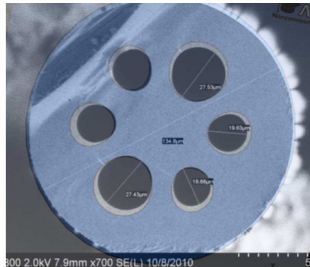
- ▶ Six holed preform, drawn at a range of fibre tensions
- ▶ Large outer diameter (30 mm), large draw ratio ( $D=45500$ )



## Multi-hole preform: Luzi et al.



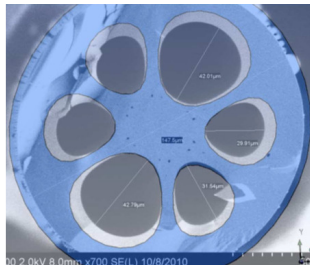
pressure=0 mbar



pressure=150 mbar

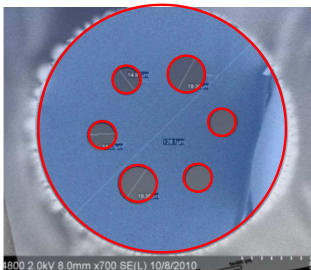


pressure=250 mbar



pressure=300 mbar

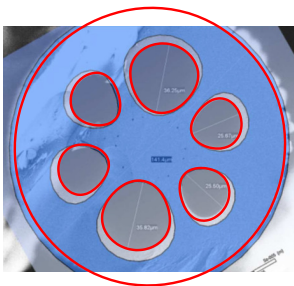
## Multi-hole preform: Luzi et al. (more)



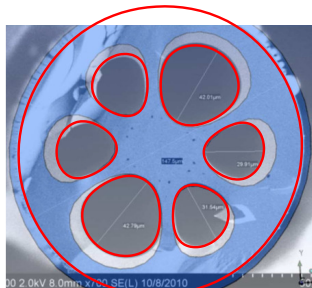
pressure=0 mbar



pressure=150 mbar



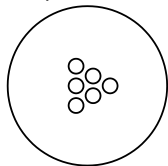
pressure=250 mbar



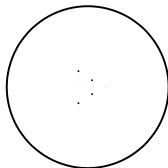
pressure=300 mbar

## Multi-hole preforms (VA preform):

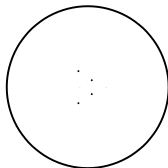
preform:



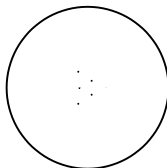
tension=15 g



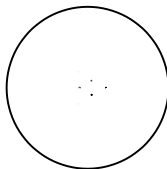
tension=19 g



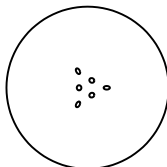
tension=27 g



tension=37 g



tension=55 g

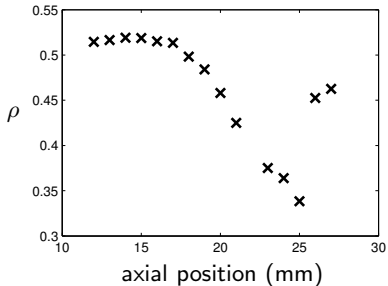
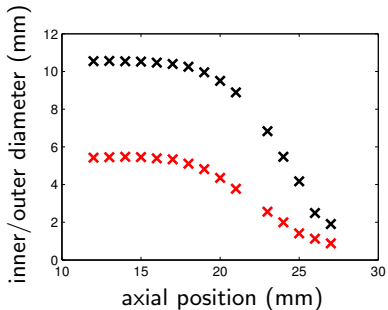


## Summary of experiments

The modelling is a good match for some of the experiments, but not others - what's going on here?

- ▶ Where they don't match it looks like the geometries are 'inflated', could this be a pressure effect?
- ▶ Discrepancies more pronounced for larger channels.

The cooled preform from experiment 4 ( $\rho_0 = 0.515$ ) was sliced up into 1mm sections.

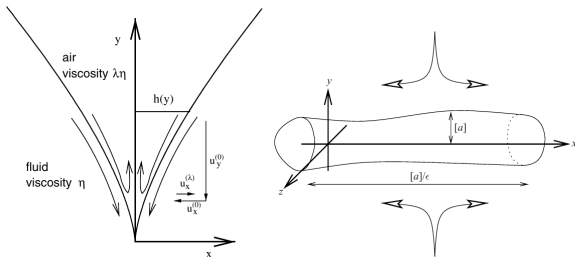


## Air entrainment in channels

There is evidence that a some kind of pressure is induced in the channels:

- ▶ Pressure 'felt' more strongly in larger channels.
- ▶ It seems to increase with tension (maybe draw speed too).
- ▶ It varies axially.

A possible explanation is that the air in the channels is entrained as the fibre is drawn (suggested by Eggers PRL '01 and Howell & Siegel JFM '04).



Suggests we need to model the flow *in* the channels.

## Air entrainment in channels (cont.)

Recall that the axial momentum equation from earlier was:

$$(3\mu^* Su_x)_x + \frac{1}{2}\gamma^* \Gamma_x + \gamma^* \sum_{k=1}^N \mathcal{P}_x^{(k)} A^{(k)} = 0$$

## Air entrainment in channels (cont.)

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Treat air as a 'weakly viscous fluid', that is  $\mu_{\text{air}} = \epsilon^2 \lambda \mu_0$ .

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Rederiving the above equation, the last term drops off to give the usual

$$\frac{d\chi}{d\tau} - \frac{\chi}{12} \tilde{\Gamma}(\tau) = -\mathcal{T}$$

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Rederiving the above equation, the last term drops off to give the usual

$$\frac{d\chi}{d\tau} - \frac{\chi}{12} \tilde{\Gamma}(\tau) = -\mathcal{T}$$

and the pressure gradient in the channels is:

$$\frac{dp_{a0}^{(j)}}{d\tau} = \frac{\mu^*(x)}{\gamma^*} \frac{\lambda}{\chi^5} \frac{1}{Q \tilde{A}^{(j)}}$$

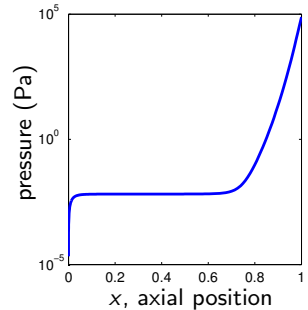
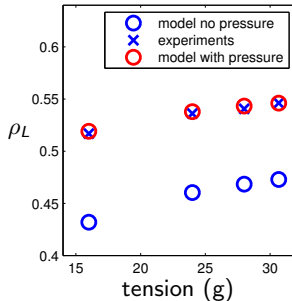
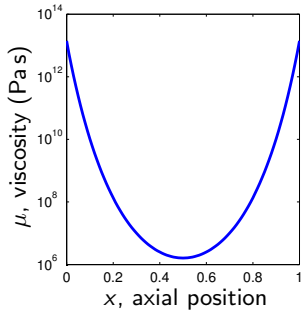
- ▶  $\tilde{A}^{(j)}$  is the channel area;
- ▶  $Q$  depends on the channel geometry, straightforward expressions for circular and elliptical cross-sections.

## Another look at the large channel tube

For a tube the pressure gradient is:

$$\frac{dp_{a0}}{d\tau} = \frac{\mu^*(x)}{\gamma^*} \frac{\lambda}{\chi^5} \frac{8\pi(1-\rho^2)}{\rho^2}$$

In the absence of knowledge about how the viscosity  $\mu^*(x)$  varies axially, choose a profile based on a Gaussian temperature profile.

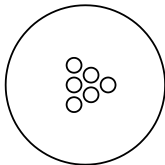


## Another look at the 6-hole preform

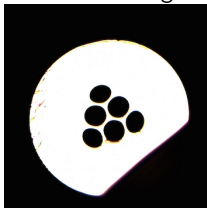
Can we get the kind of inflation we saw in the experiments?

Choose a (slightly) less steep viscosity curve:

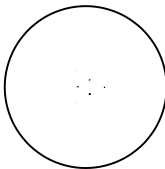
preform:



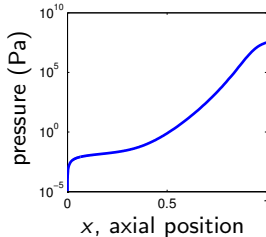
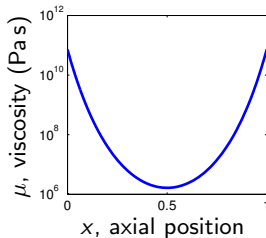
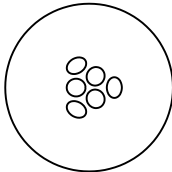
tension=37 g



no pressure:



pressure:



Reasonable match for the geometry - pressure quite large for this case.

## Future work

- ▶ Extend to non-elliptical geometries.
- ▶ Modelling of the glass temperature, to get more realistic viscosity profiles.

Thanks!



Some papers:

Chen et al., *Asymptotic Modelling of a Six-Hole MOF*, J. Lightwave Tech. 34(24) 2016.

Buchak & Crowdy, *Surface-tension-driven Stokes flow: A numerical method based on conformal geometry*, JCP 317, 2016.