

DNS of a turbulent jet impinging on a heated wall

Thibault Dairay

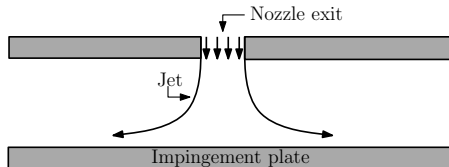
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24/04/2014

Impinging jet

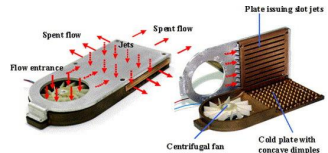
- A fluid issuing from a nozzle impinging on a solid wall
- Efficient tool to optimize flow/wall **heat transfer**
- Widely used in industrial applications requiring **high transfer rates** (cooling, drying...)



Some applications

Cooling of electronic components

- Increase of computing ressource requirements
- **Efficient and simple** solution



Chang et al., ETFS, 2007

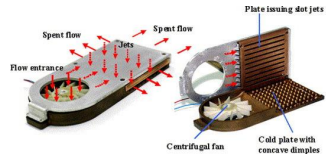
Vertical Take-Off and Landing aircraft

- **Vertical force** generated by impinging jets
- Control the jets : improve the aircraft stability

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Vertical Take-Off and Landing aircraft

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Take-off of an AV8B-Harrier ©2013

ForWallpaper.com

Cooling of turbine blades

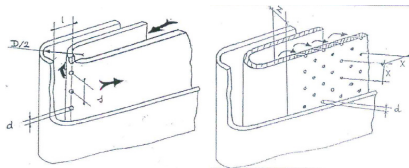
Industrial challenge

Improve aircraft engine's cooling is a major challenge for the manufacturers

- Optimize the engine **efficiency**
- Increase its **life duration**
- Decrease the maintenance **cost**

Different cooling technics

- Forced internal convection cooling
- Film cooling
- **Impinging jet cooling**



Fénot, PhD Thesis, 2004

Cooling of turbine blades

At the lab

- Many experimental research at the Institute PPRIME
- Industrial collaboration : DGA, Snecma Moteurs, Turboméca, ONERA



PhD Thesis : Bernard (1997); Jolly (2000); Brevet (2001); Fénot (2004); Roux (2011)

Industrial system

Physical analysis : single jet

Cooling of turbine blades

At the lab

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Industrial system

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At the lab

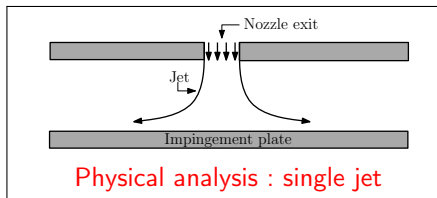
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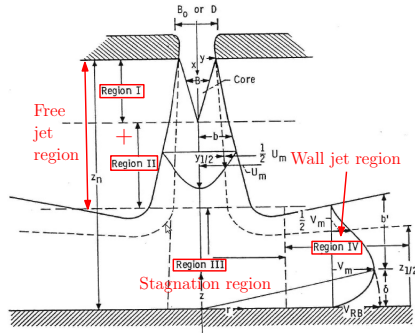


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Industrial system



Single impinging jet

- A boundary-free shear flow region
- A **wall-bounded** flow region
- A **stagnation region**
- Accelerated/decelerated **wall jet**



Adapted from



Gauntner *et al.*, TR-NASA, 1970

Single impinging jet : multiple parameters

A lot of possible configurations depending on the choice of **geometrical**, **aerodynamical** or **thermal** parameters :

- Jet type, nozzle-to-plate distance...
- Inflow velocity, Reynolds number...
- Plate heating, temperature difference between the plate, the jet and the surrounding fluid...

Experimental studies



Review papers of Jambunathan *et al.*, IJHFF, 1992 ; Webb and Ma, AHT, 1995



PhD thesis of Vejrazka (2002) and Roux (2011)

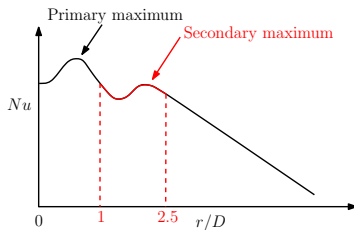
Numerical studies



Hadziabdic and Hanjalic, JFM, 2008 ;
Dewan *et al.*, HTE, 2012 ;
Uddin *et al.*, IJHMT, 2013

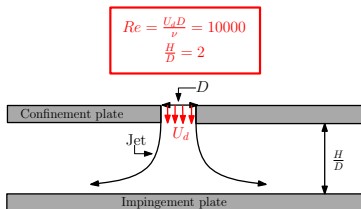
Today's motivations

- **Secondary maximum** in the radial distribution of the mean Nusselt number
- **Measurement technics limitations** : synchronized temporal evolution of the 3D fields is extremely difficult to obtain, measurement of the near-wall dynamics
- Unsteady numerical studies ($Re \geq 10000$) : exclusively **Large Eddy Simulation (LES)**



Today's study

DNS of a circular confined jet impinging on a flat plate in a realistic flow regime



What is the role of unsteady processes to explain the spatial distribution of the mean heat transfer at the wall ?

For experimental validation (PIV, IR Thermography) and comparisons with different LES methodologies, see



Dairay et al. under review, IJHFF 2014

Contents

- 1 Numerical methods
- 2 Main turbulent statistics
- 3 Unsteady processes and heat transfer distribution at the wall
- 4 Conclusion

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Problem equations

Incompressible Navier-Stokes equations

$$\frac{\partial \mathbf{u}}{\partial t} + \frac{1}{2} (\nabla \cdot (\mathbf{u} \otimes \mathbf{u}) + (\mathbf{u} \cdot \nabla) \mathbf{u}) = -\frac{1}{\rho} \nabla p + \nu \nabla^2 \mathbf{u}$$

$$\nabla \cdot \mathbf{u} = 0$$

$$\text{Reynolds number : } Re = \frac{U_d D}{\nu}$$

Temperature equation (passive scalar)

$$\frac{\partial T}{\partial t} + \mathbf{u} \cdot \nabla T = \frac{1}{RePr} \nabla^2 T$$

$$\text{Prandtl number : } Pr = \frac{\nu}{\kappa}$$

Incompact3d

Solve the **incompressible Navier-Stokes equations** on a Cartesian mesh.

- Spatial discretization : **High-order compact schemes** $O(\Delta x^6)$
- Time advancement : **Explicit** Adams-Bashforth scheme coupled with an **implicit** Crank-Nicolson scheme $O(\Delta t^2)$
- Pressure treatment in the **spectral space** with the help of the modified wave number concept
- **Parallel version** (MPI implementation and pencil domain decomposition)



S. Laizet and E. Lamballais, JCP, 2009

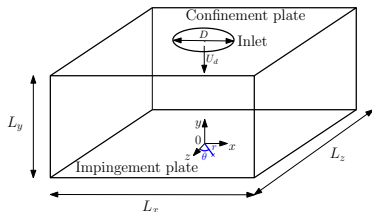


S. Laizet *et al.*, CF, 2010



S. Laizet and N. Li, IJNMF, 2011

Computational domain and boundary conditions



- **Physical parameters** : $Re = 10000$, $H/D = 2$, $Pr = 1$

- **Numerical parameters** :

- $n_x = n_z = 1541 \Rightarrow \Delta x^+ \approx 10$
- $n_y = 401 \Rightarrow 0.9 \lesssim \Delta y^+ \lesssim 40$
- $\Delta t = 2.10^{-4}$



T. Dairay *et al.*, ETC14, 2013

Velocity

- Inflow : Mean velocity prescribed (power law) + synthetic perturbations
- Outflow : Buffer zone
- Top and bottom plates : No-slip condition

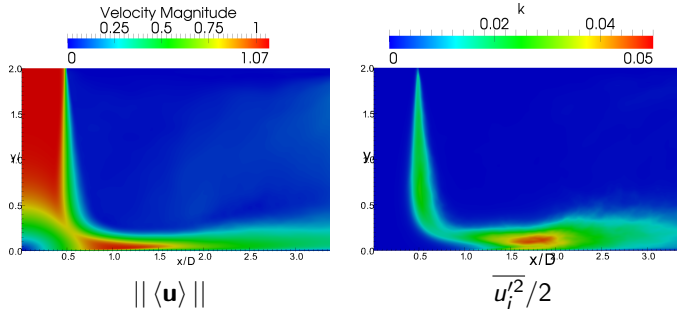
Temperature

- Inflow : Constant temperature T_j
- Outflow : Convective condition
- Top plate : Constant temperature T_j
- Bottom plate : Constant heat flux φ_p

Contents

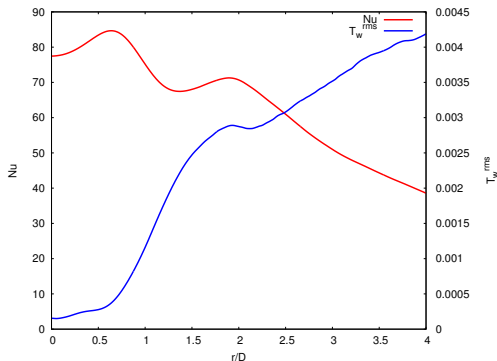
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Velocity statistics



- Free jet region ($0.5 < y/D < 2$) : Potential core, low turbulence intensity on the jet axis ($u_y = U_{max}$), shear layer $r/D \approx 0.5$
- Stagnation region ($0 < y/D < 0.5$ and $r/D < 1.8$) : Axial to radial flow, boundary layer from the stagnation point, u_r increase, k maximum
- Wall jet region ($r/D > 1.8$) : radial velocity, u_r and k decrease

Temperature statistics

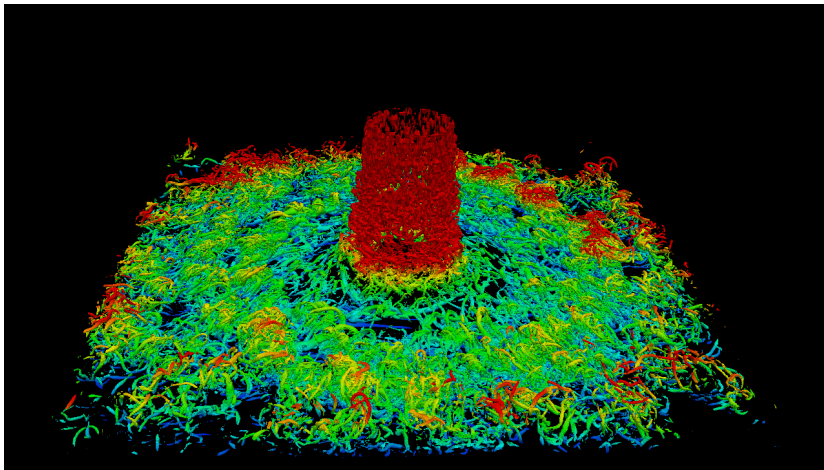


- **Local minimum at the stagnation point** : low turbulence level on the jet axis (potential core)
- **Primary maximum at $r/D \approx 0.7$** : structures issuing from the jet shear layer
- **Secondary maximum at $r/D \approx 2$** : linked with the region $1 \leq r/D \leq 2$ of high k

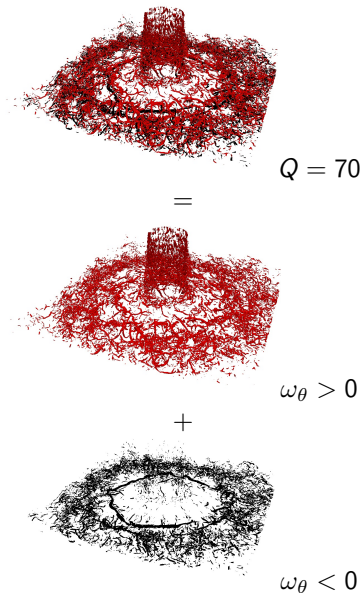
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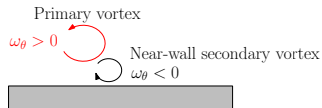
Flow animation : Q criterion colored by y/D



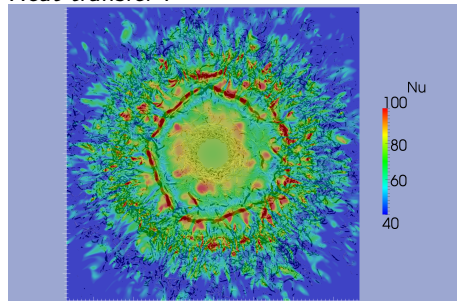
Large-scale organisation



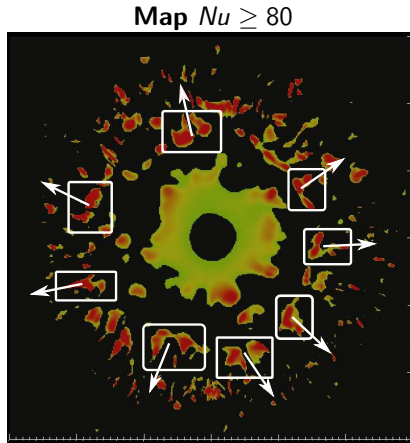
2D toroidal structures :



Heat transfer :



Small-scale organisation



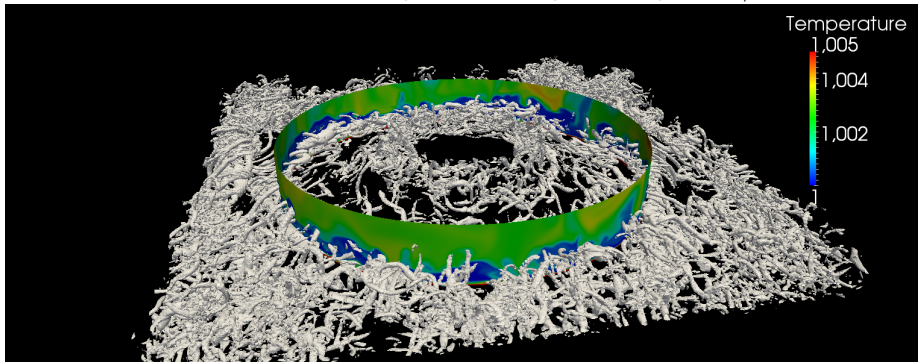
- Cold spots : extremely high heat transfer
- Radial stretching : filament propagation of cold spots



Hadziabdic et Hanjalic, JFM, 2008; Uddin et al., IJHMT, 2013

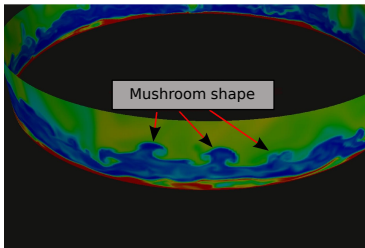
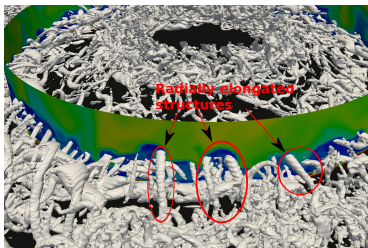
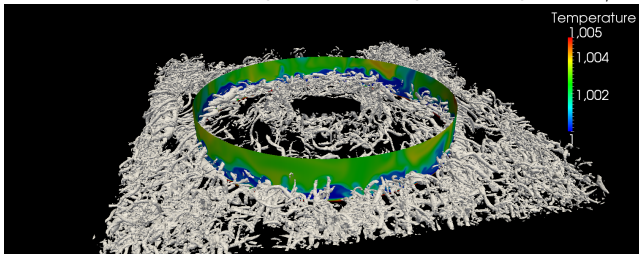
Small-scale organisation

Isosurface $Q = 50$ and temperature map in the plane $r/D = 1.5$

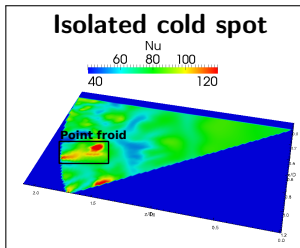


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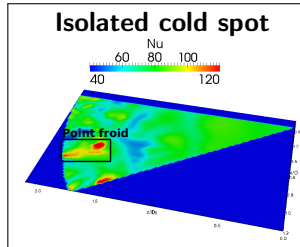
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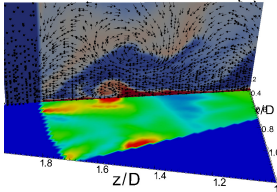
Instantaneous fields in the vicinity of a cold spot



Instantaneous fields in the vicinity of a cold spot

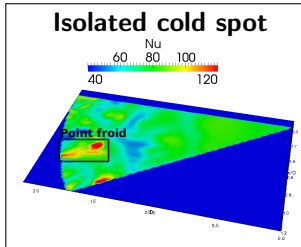


T in the plane $\theta = Cste$

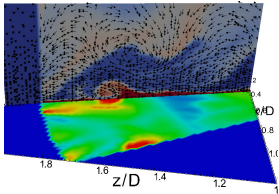


2D toroidal

Instantaneous fields in the vicinity of a cold spot

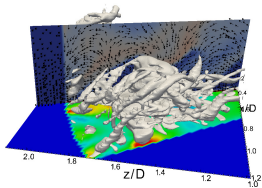


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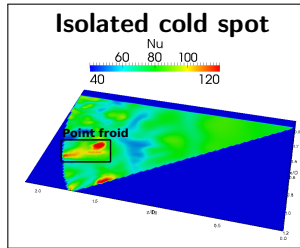


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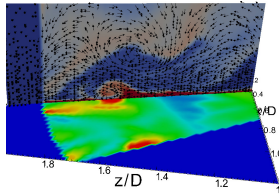
Isosurface $Q = 10$



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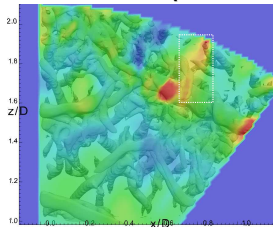


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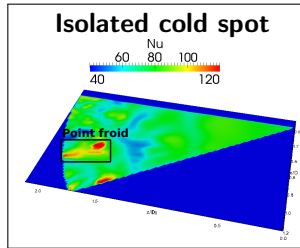
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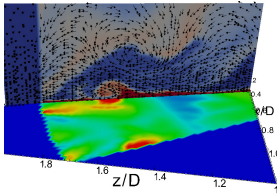


Radial structures

Instantaneous fields in the vicinity of a cold spot

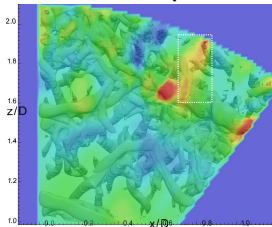


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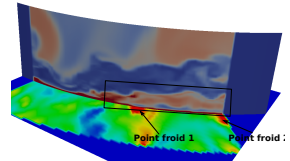
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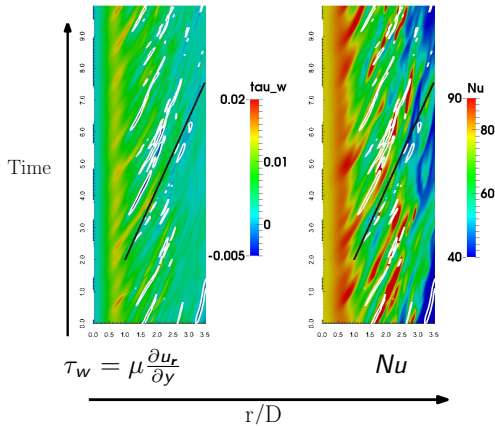
Radial structures

T in the plane $r/D = 1.77$



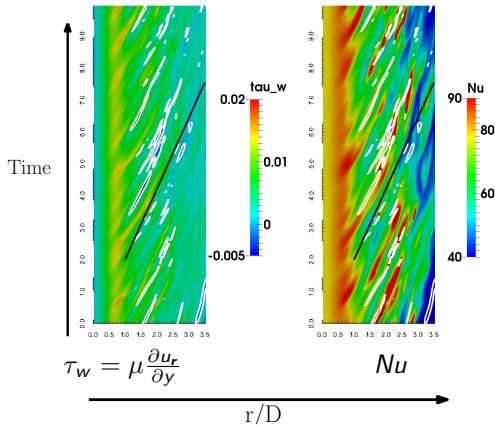
Azimuthal distortion

Spatio-temporal maps



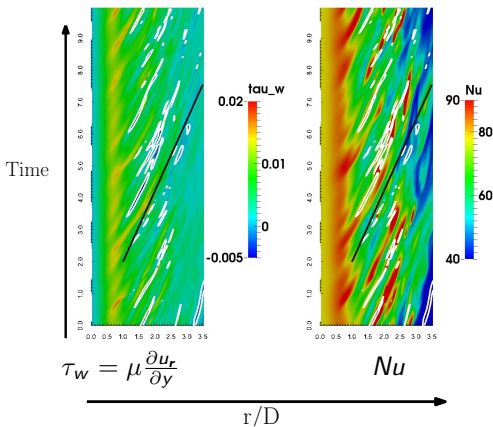
- For $r/D < 0.5$: steady evolution \Rightarrow **local minimum** of \overline{Nu}
- For $0.5 < r/D < 1$: periodical appearance of high heat transfer values \Rightarrow **primary maximum** of \overline{Nu}
- For $1 < r/D < 2.5$: secondary vortex and cold fronts convected at $U_c \approx 0.45 U_d \Rightarrow$ **increase** of \overline{Nu}

Spatio-temporal maps



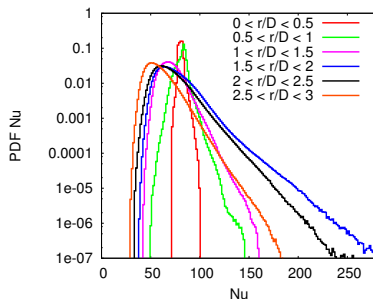
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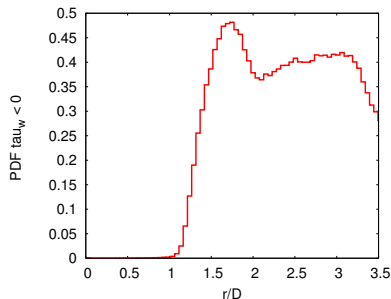
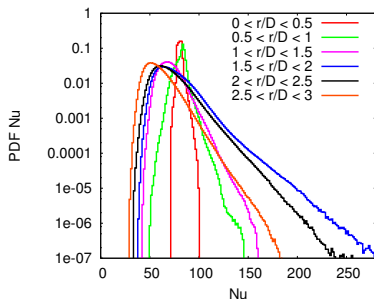
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Probability Density Functions



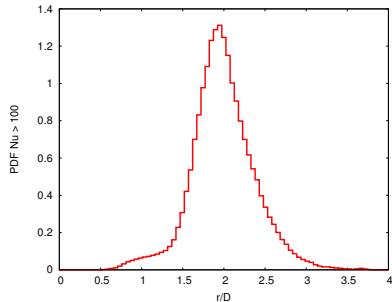
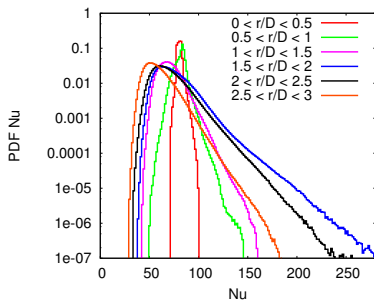
- For $1.5 \leq r/D \leq 2.5$ (blue and black curves) : **highly skewed PDF** with $Nu > 200 \implies$ **rare but extremely intense** thermal events
- Most probable location of $\tau_w \leq 0$ in the range $1.5 \leq r/D \leq 2$
- Extreme events localised in the range $1.5 \leq r/D \leq 2.5$ with a maximum probability at $r/D \approx 2$

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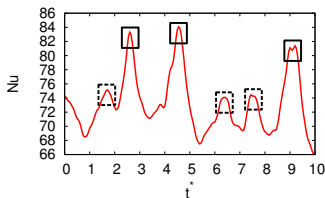
Cold spots detection and conditional averaging

Intermediate radial location $r/D = 1.75$

- 1 Selection of comparable instants
- 2 Detection of the azimuthal angles corresponding to a cold spot (threshold $S = 110$)
- 3 Conditional averaging operator $\langle f(r, \xi, y, t_0) \rangle_{Nu} = \frac{1}{N} \sum_{j=1}^N f(r, \theta_j + \xi, y, t_0)$
- 4 Temporal averaging of each conditionally averaged field

Temporal evolution of the azimuthally averaged

Nusselt number at $r/D = 1.75$



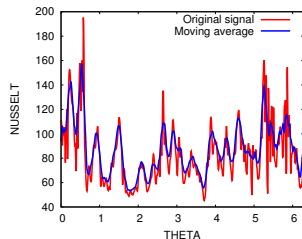
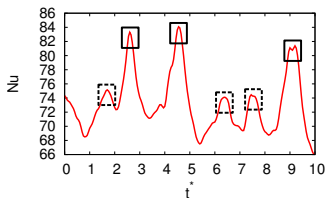
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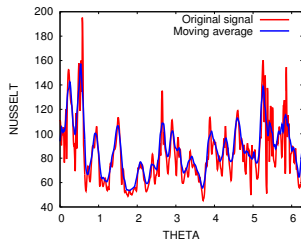
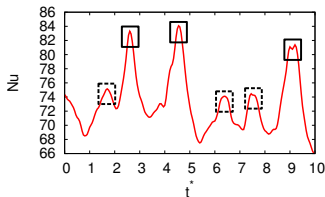
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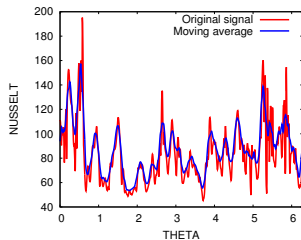
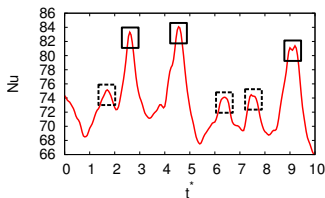
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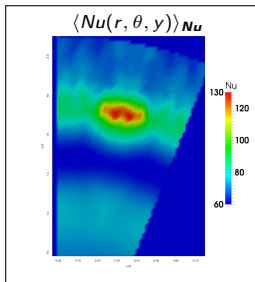
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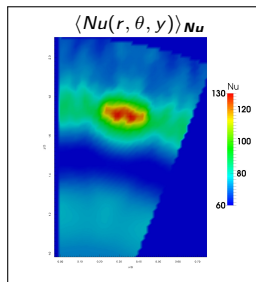
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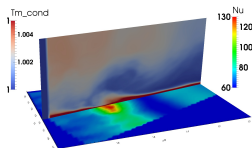
Aerothermal fields in the vicinity of a cold spot



Aerothermal fields in the vicinity of a cold spot

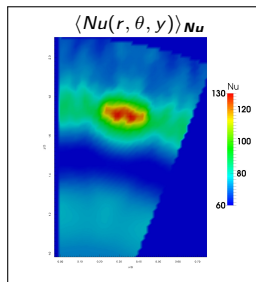


$\langle Nu(r, \theta, y) \rangle_{Nu}$ and $\langle T(r, \theta, y) \rangle_{Nu}$
in a plane $\theta = cste$

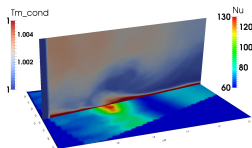


Cold fluid flux

Aerothermal fields in the vicinity of a cold spot

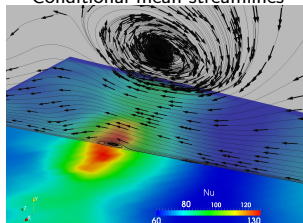


$\langle Nu(r, \theta, y) \rangle_{Nu}$ and $\langle T(r, \theta, y) \rangle_{Nu}$
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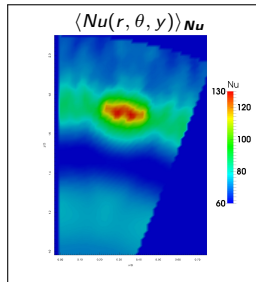
Cold fluid flux

Conditional mean streamlines

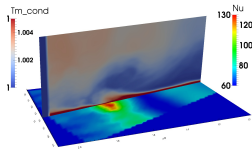


Primary and secondary vortices

Aerothermal fields in the vicinity of a cold spot

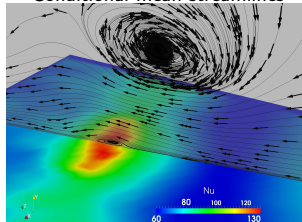


$\langle Nu(r, \theta, y) \rangle_{Nu}$ and $\langle T(r, \theta, y) \rangle_{Nu}$
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Cold fluid flux

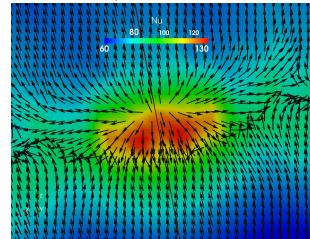
Conditional mean streamlines



Primary and secondary vortices

Conditional mean streamlines
projected in the near-wall plane

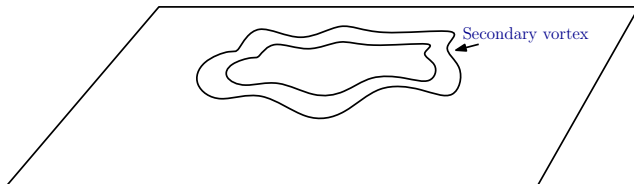
$y/D = \Delta y_{min}$



Contents

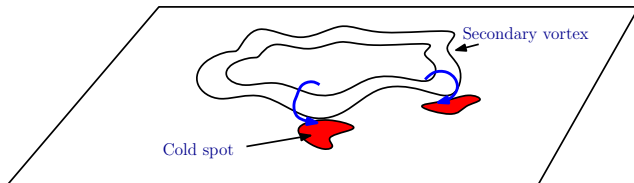
- 1 Numerical methods
 - Numerical tool
 - Flow configuration
- 2 Main turbulent statistics
- 3 Unsteady processes and heat transfer distribution at the wall
 - Instantaneous visualisations
 - Temperature distribution analysis
 - Conditional averaging of temperature and velocity
- 4 Conclusion

Unsteady processes and heat transfer distribution at the wall



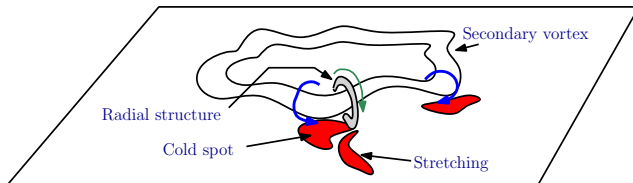
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 - **Large-scale** toroidal organisation
 - Azimuthal distortion → cold spots
 - Radial stretching
- Spatio-temporal maps : cold fronts convected at the same velocity as the secondary vortex
- PDF : extremely high heat transfer linked with cold spots occurrence in the vicinity of the secondary maximum ($1.5 \leq r/D \leq 2.5$)
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Unsteady processes and heat transfer distribution at the wall



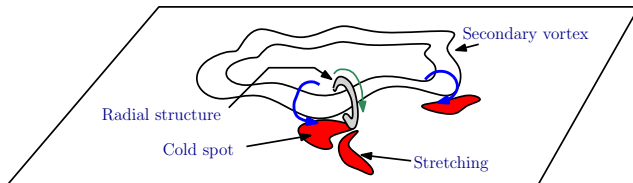
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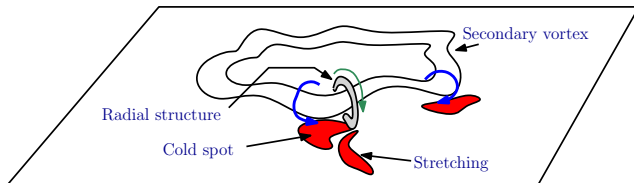
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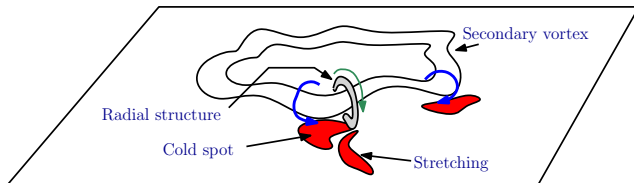
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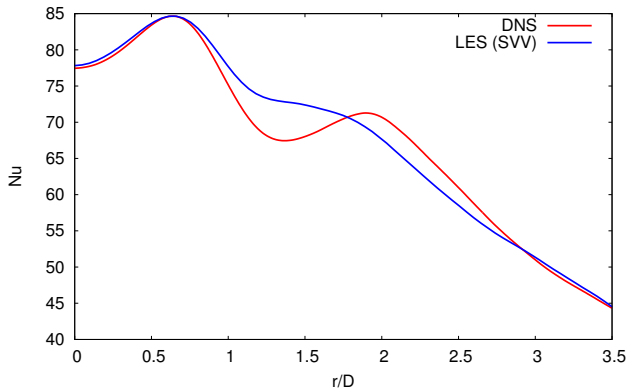
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Additional comment

The secondary maximum amplitude is reduced when using **LES approaches** → confirm the importance of the **small-scale contribution** to the mean heat transfer



Thank you